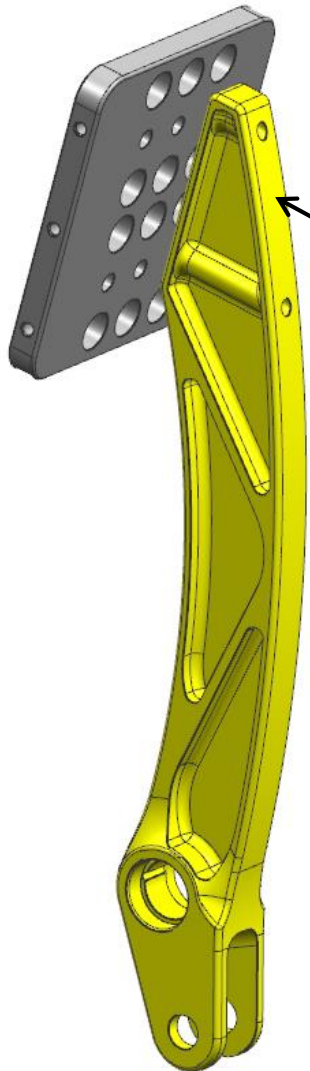


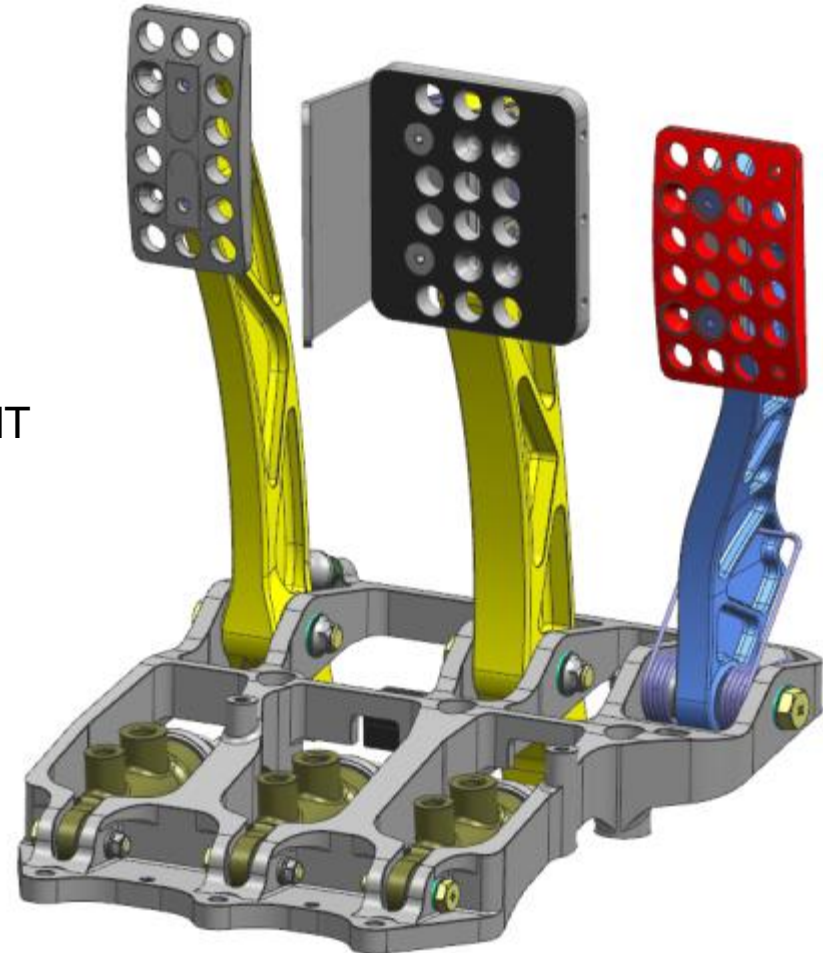
LATTICE STYLE OPTIMIZATION STUDY RACE CAR BRAKE PEDAL

PRESENTED BY: GARY LATHAM
PRATT & MILLER ENGINEERING

RACE CAR PEDALS USED IN CHAMPIONSHIP
WINNING CORVETTE, CADILLAC AND CAMARO



NOTICE HOW PAD
FACE IS OFFSET ON
THE ARM THUS
CREATING A MOMENT
ABOUT THE Z AXIS



10 STEP GUIDE TO CREATING LATTICE OPTIMIZED PARTS

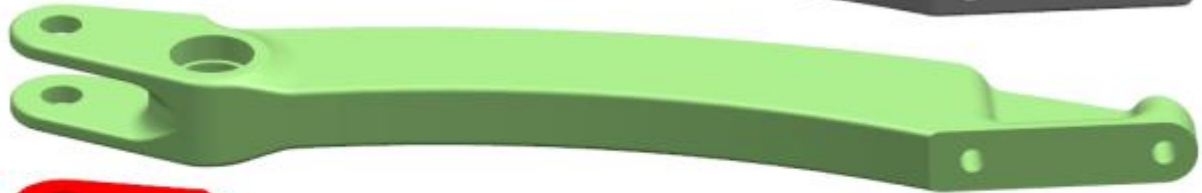
- 1. CREATE DESIGN SPACE MODEL AND LOAD CASES – JUST LIKE A STANDARD OPTIMIZATION**
- 2. USE A COURSE GRID SIZE**
- 3. START WITH A SLIGHTLY SMALLER VOLUME TO ACCOUNT FOR BEAM GROWTH**
- 4. IDENTIFY UPPER AND LOWER BOUNDS FOR OPTIMIZATION AND BEAM SIZE**
- 5. OPTIMIZATION RESULTS ARE THEN TRANSFERRED TO MATERIALISE FOR STL CREATION**
- 6. CREATE BEAM STL – NOTICE SIZE IMPACT ON REFINEMENT**
- 7. CREATE SURFACE WRAP OF SOLIDS WITH INFLATION**
- 8. CREATE ADDITIONAL FILL AREAS AND MERGE ALL TOGETHER**
- 9. IMPORT FINAL BUILD SOLID AND INTERSECT WITH ABOVE**
- 10. PRINT IT!**

	Mass	Displacement	Percent Weight Delta From Full Volume	Displacement Ratio From Full Volume	Percent Weight Delta From Current	Displacement Ratio From Current
Full Volume	423.1	0.744	100.0%	100.0%		
Webbed Pedal	226.3	3.112	53.5%	29.1%		
Lattic Optimized	218.0	2.217	51.5%	44.7%	3.7%	128.8%

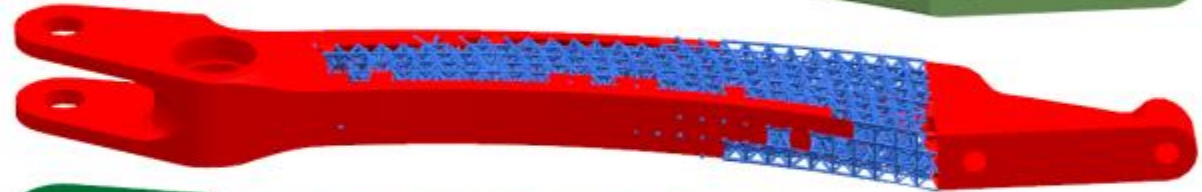
CURRENT MACHINED PEDAL



DESIGN SPACE AVAILABLE



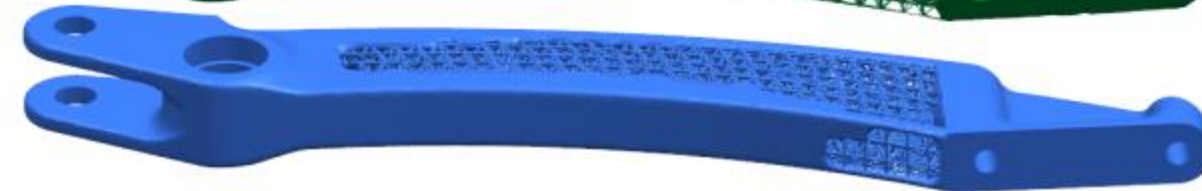
RAW OPTIMIZED PEDAL

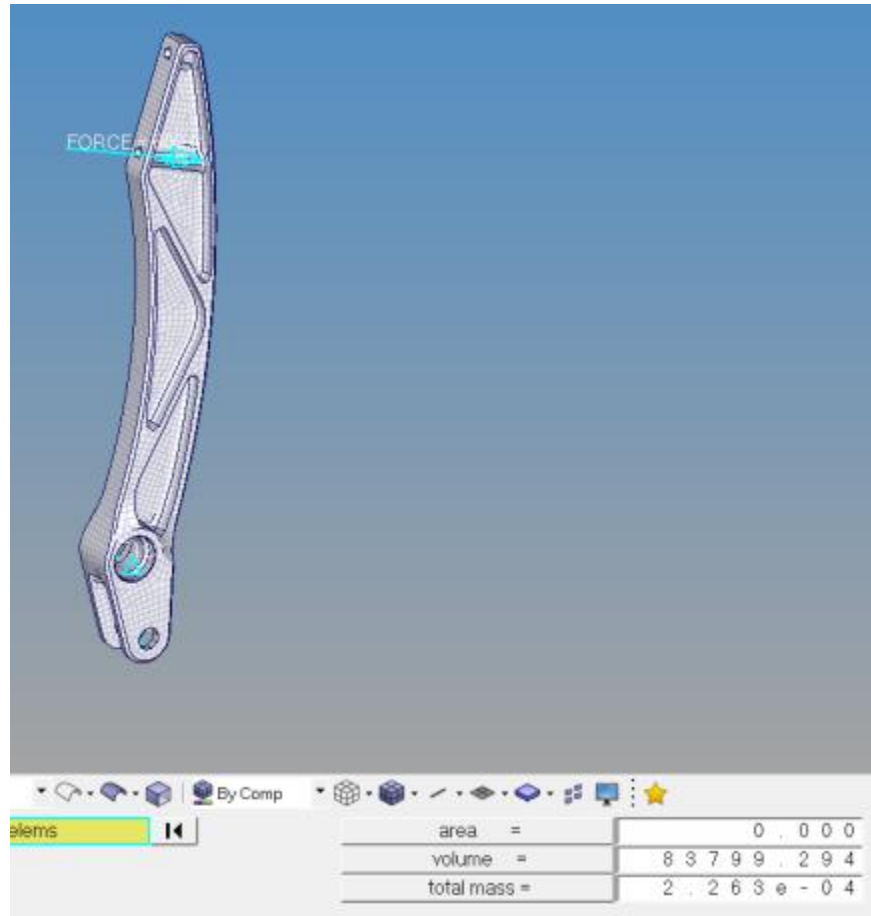


SMOOTHED STL PEDAL



FINAL LATTICE DESIGN

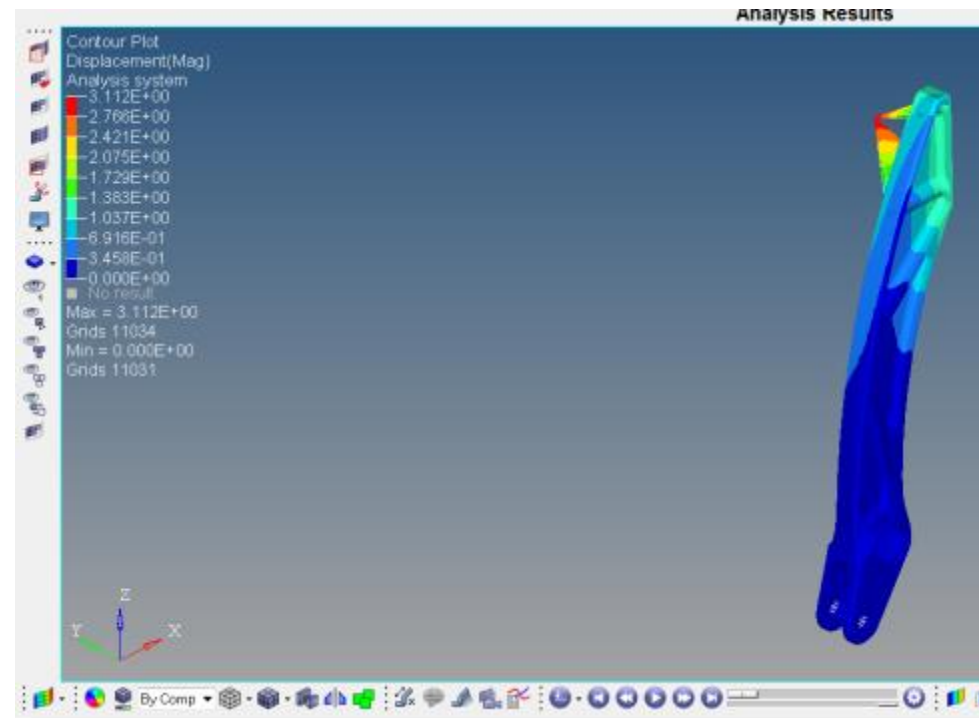




CURRENT DESIGN SPECS

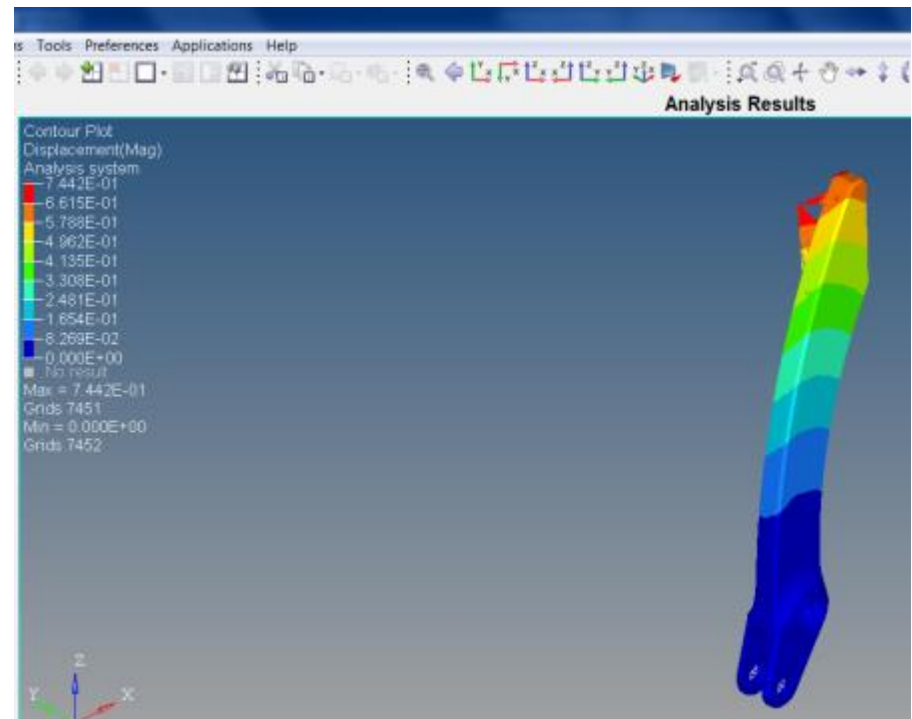
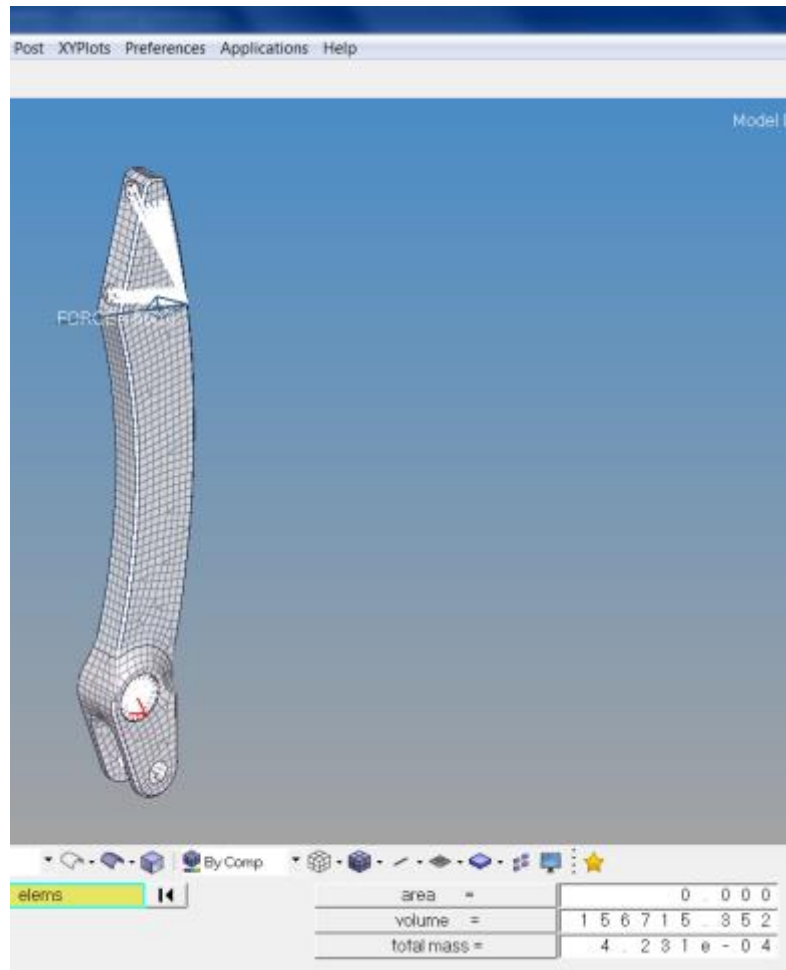
226.3 g

3.11 mm



SPACE CLAIM MAXIMUMS

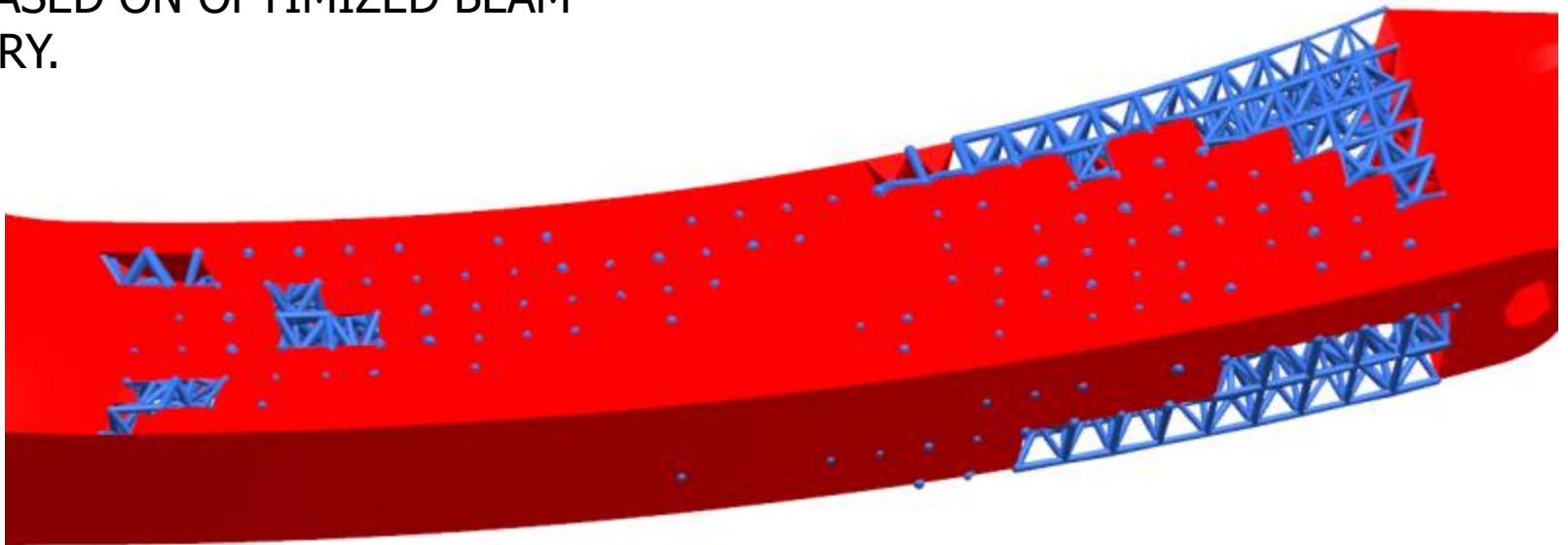
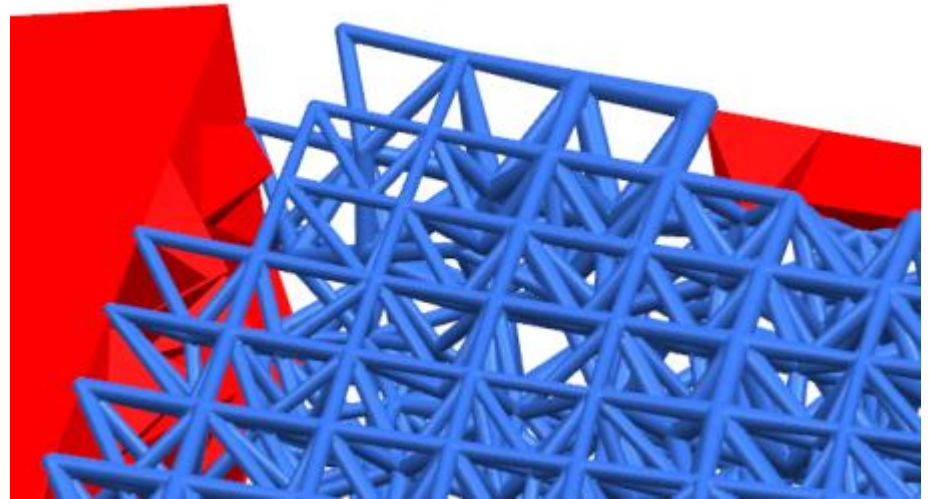
423.1 g
0.744 mm



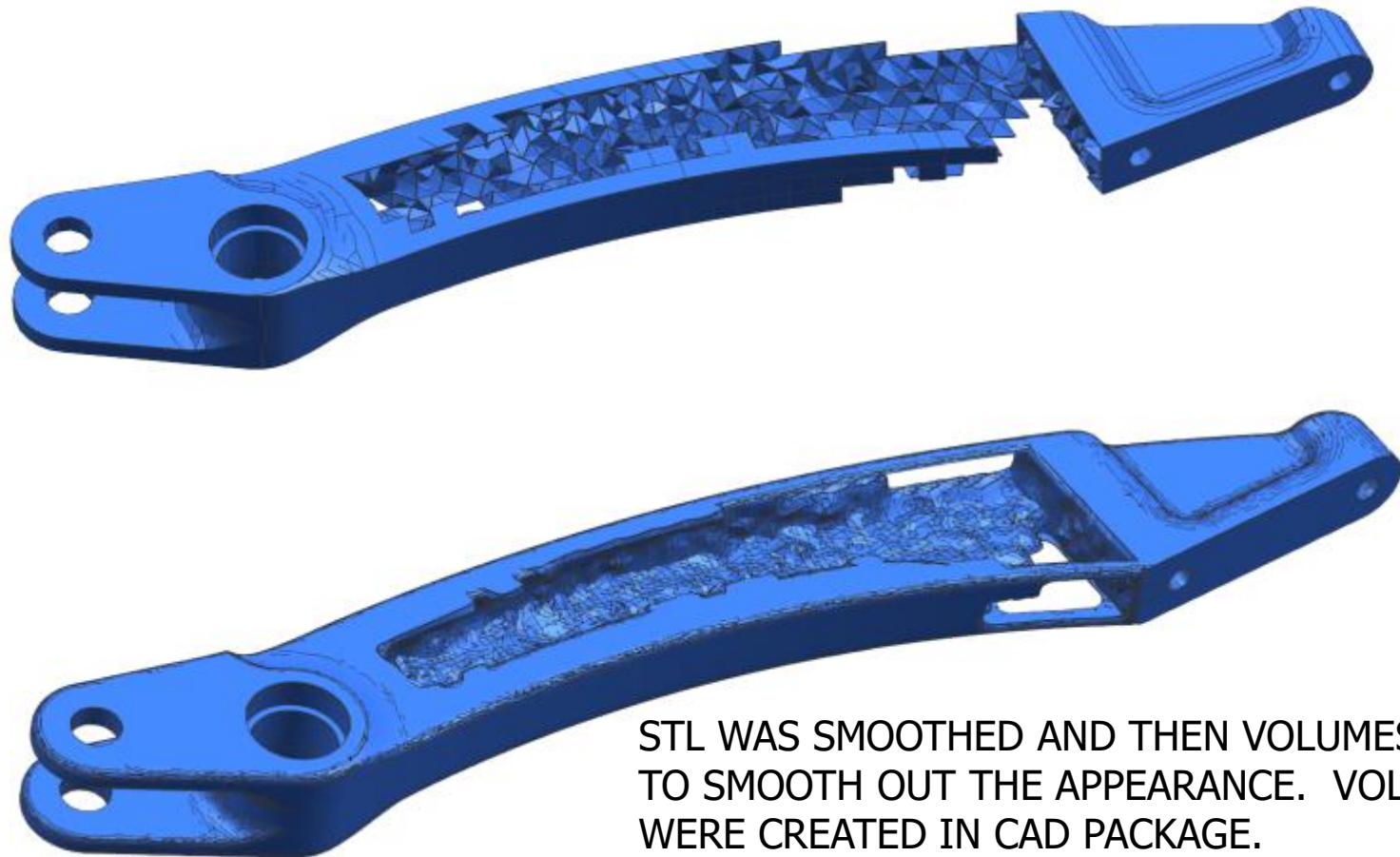
LATTICE STRUCTURE AFTER STL
CREATION IN MATERIALISE.

NOTICE HOW THE BEAMS ARE NOW
OUTSIDE THE ORIGINAL PART
SURFACE DUE TO BEAM INFLATION.

ALSO NOTICE HOW THE BEAMS
TAPER BASED ON OPTIMIZED BEAM
GEOMETRY.

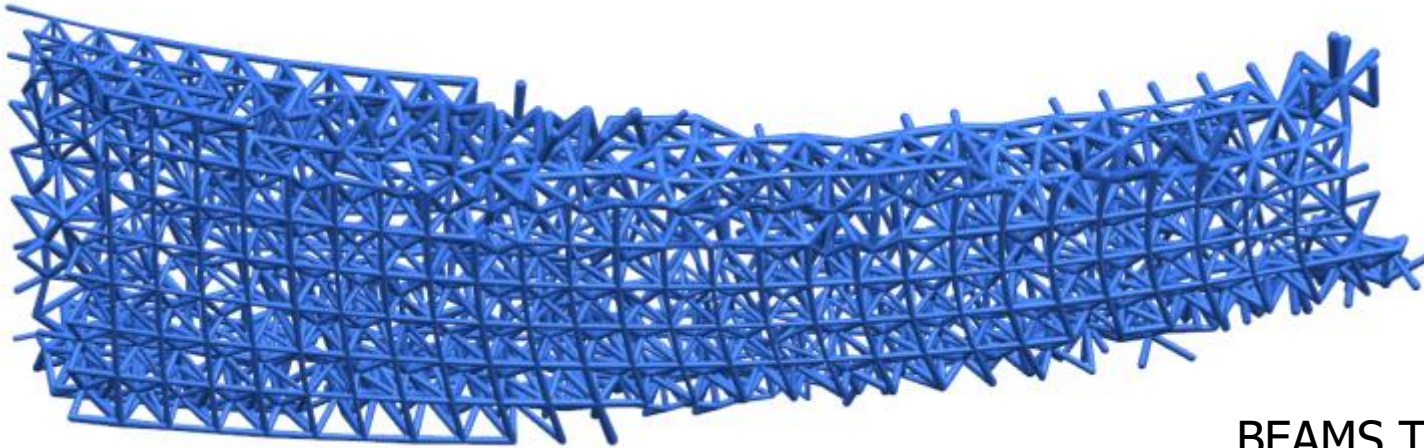


TETRAHEDRAL FEA AREA AND THE FINAL SOLID AREA

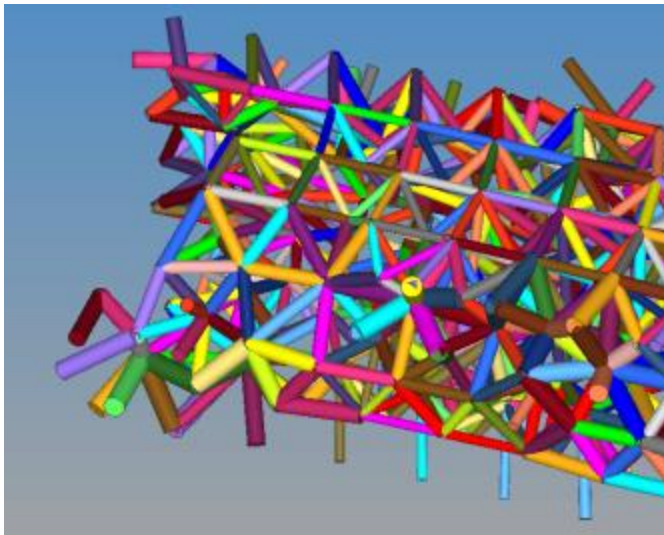


STL WAS SMOOTHED AND THEN VOLUMES ADDED TO SMOOTH OUT THE APPEARANCE. VOLUMES WERE CREATED IN CAD PACKAGE.

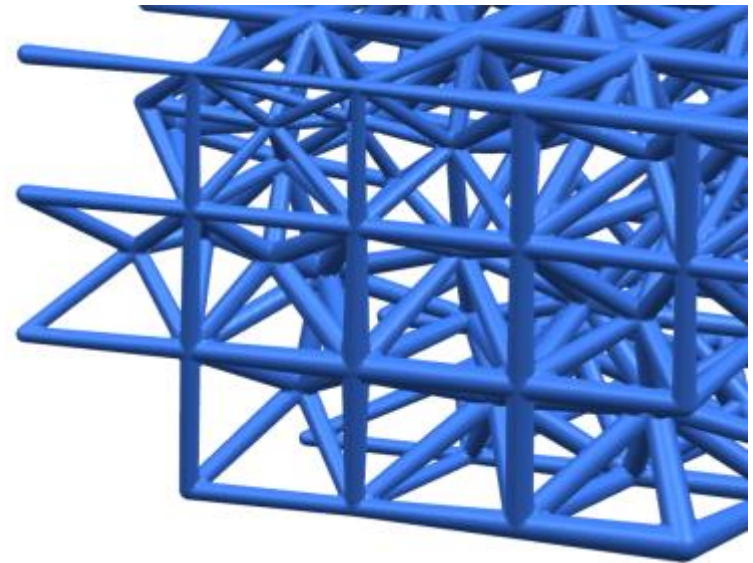
BEAM ELEMENTS AFTER STL CREATION



OPTIMIZED BEAMS IN OPTISTRUCT

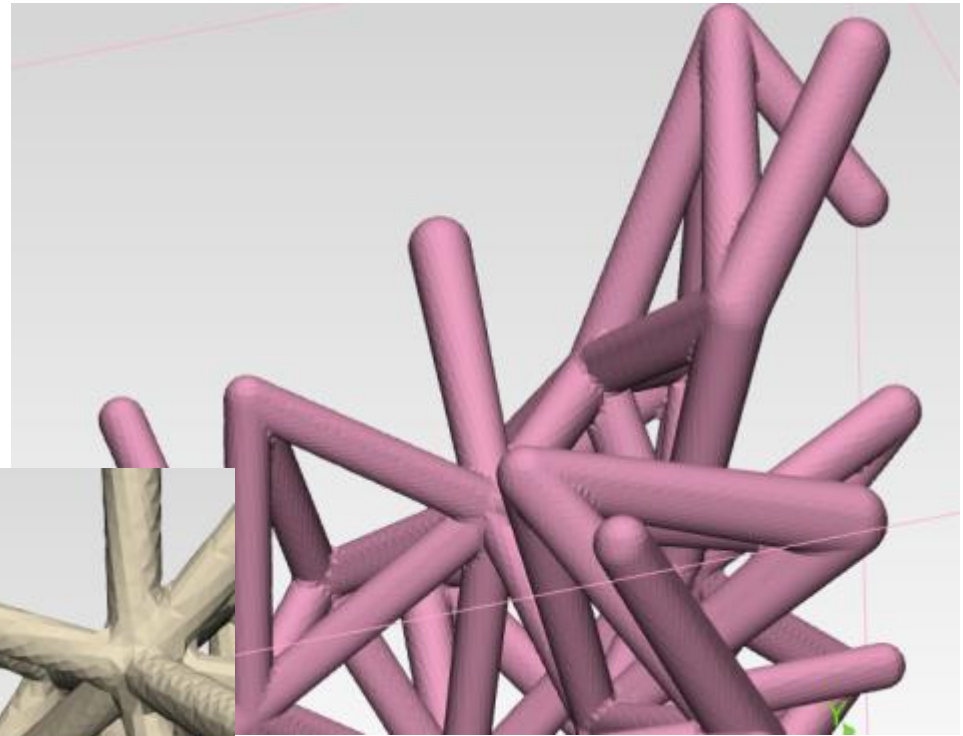
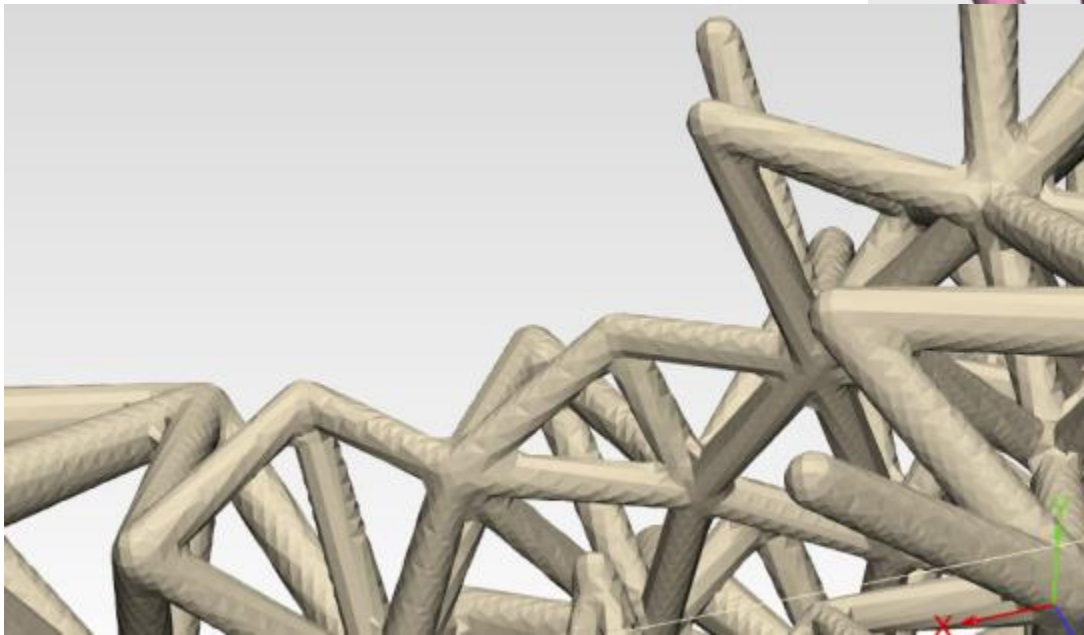


BEAMS TAPERED AND VARIABLE THICKNESS



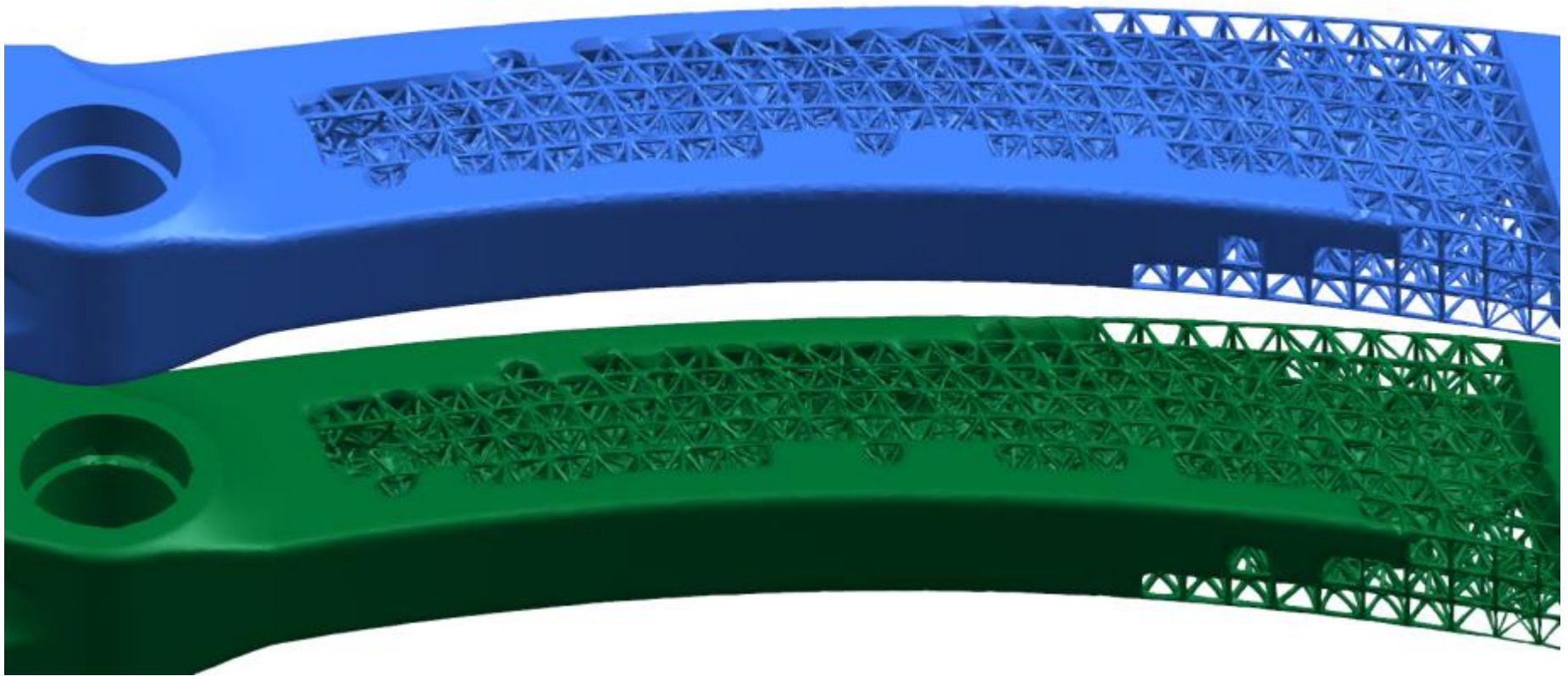
FACETING RESOLUTION ON
LATTICE BEAMS.

FINE MESH IS SET TO 0.1
COURSE MESH IS SET TO 0.2



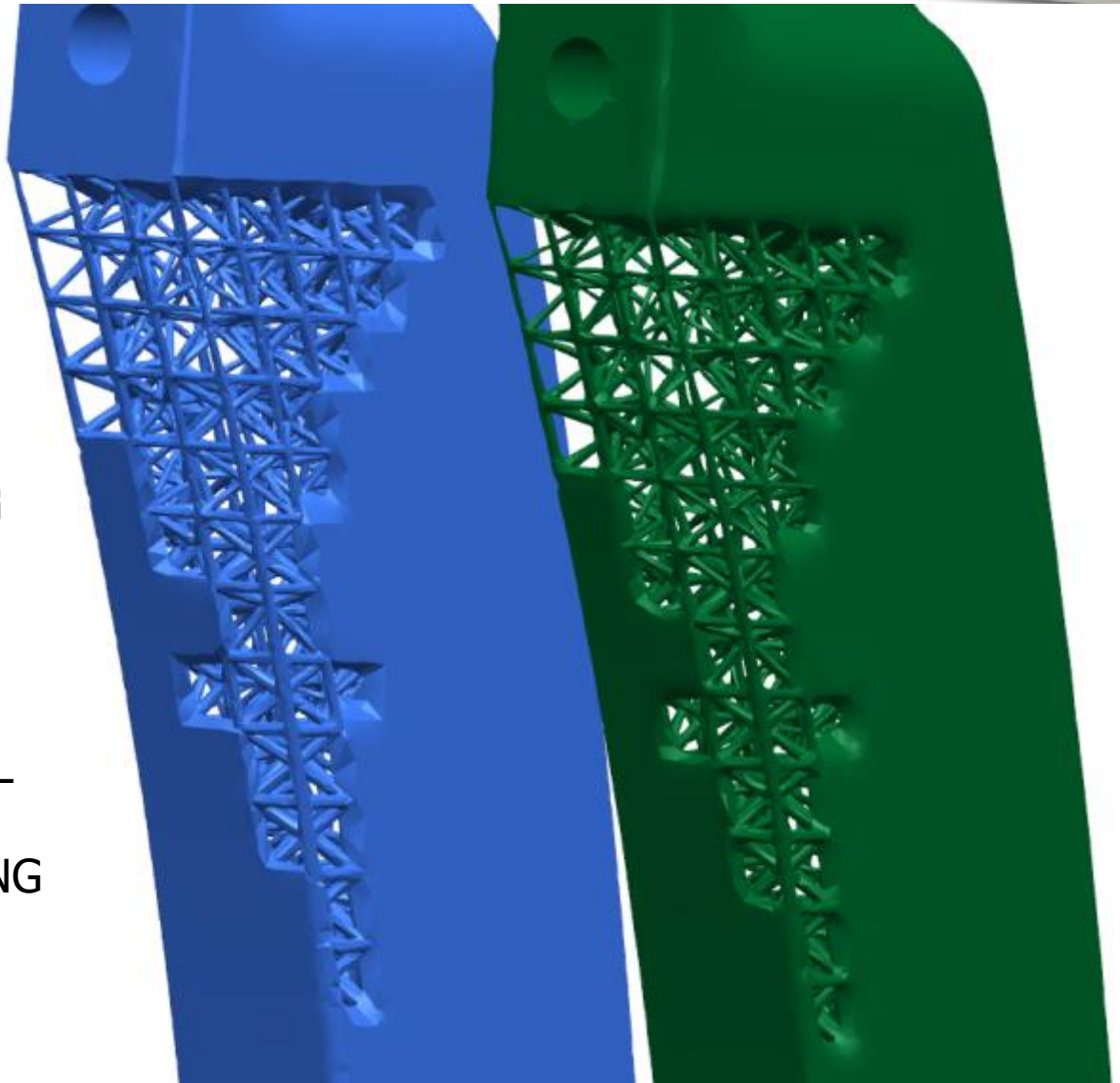
SMOOTHING THE RESULT FILE

PART LOOKS BETTER BUT ALSO HAS SOME ERRORS CREATED IN THE BEARING BORES. WHEN TO SMOOTH YOUR PART IS CRITICAL AND WHEN TO INTERSECT THE PART WITH YOUR DESIRED FINAL VOLUME IS ALSO CRITICAL



ADDITIONAL SMOOTHING

EVEN WITH ADDITIONAL SMOOTHING, THE PART STILL LOOKS TOO DISCRETIZED. SOME OPEN EDGE SMOOTHING WOULD BE USEFULL.



EDGES OF SOLID ELEMENTS DID NOT LOOK PROPER AND THE BEAM END STUCK PAST SOME OF THE SURFACES.

A VOLUME WAS CREATED IN CAD THAT OFFSET THE OUTER SURFACE BY 0.75MM AND RETRIMMED THE OPEN FACES FOR A MORE AGREEABLE EDGE.

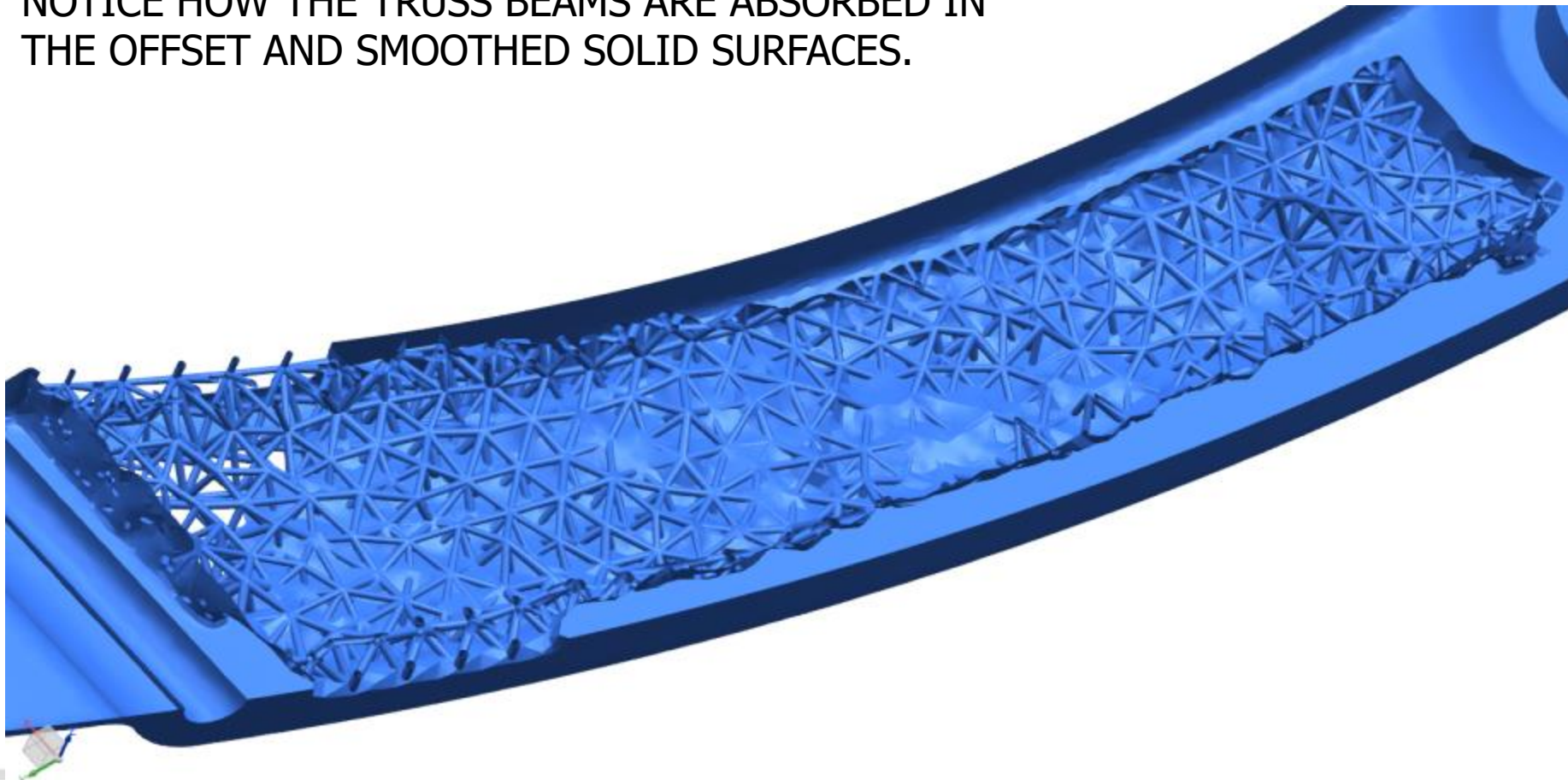


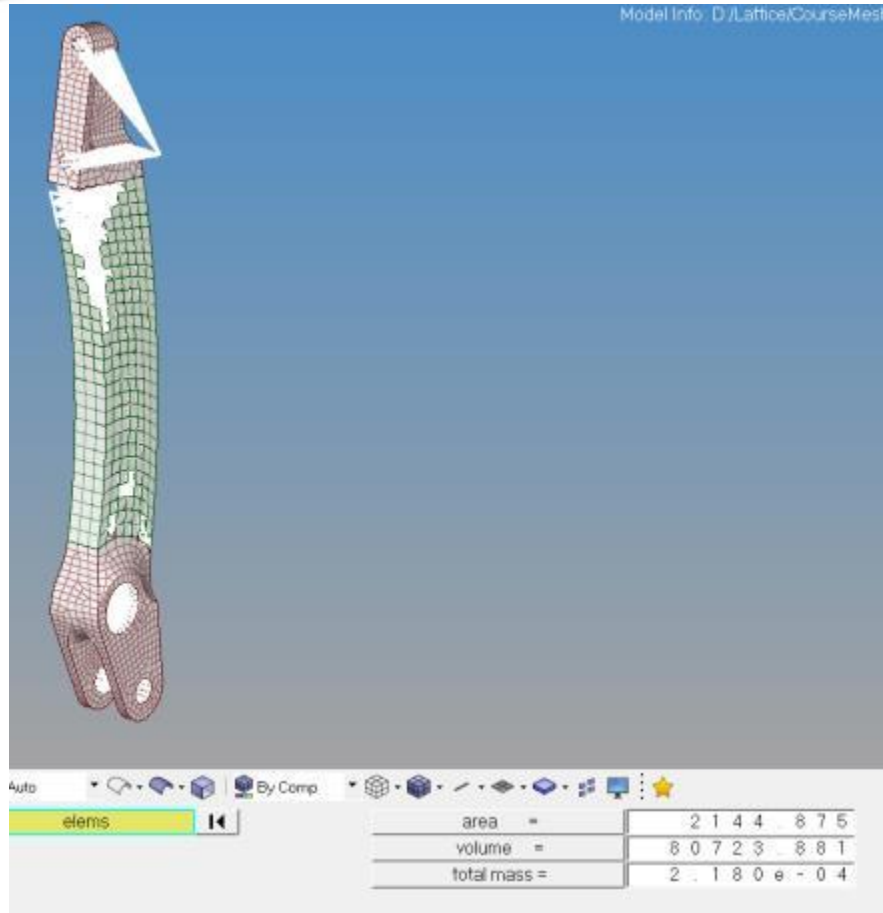
FINAL PART SHOWN. THE OPENINGS COULD BE COMPLETELY SMOOTHED IF DESIRED. THE MATERIAL ISN'T REQUIRED FOR THE PART BUT MAKES IT "LOOK" MORE COMPLETE".



INTERIOR OF MESH

NOTICE HOW THE TRUSS BEAMS ARE ABSORBED IN THE OFFSET AND SMOOTHED SOLID SURFACES.





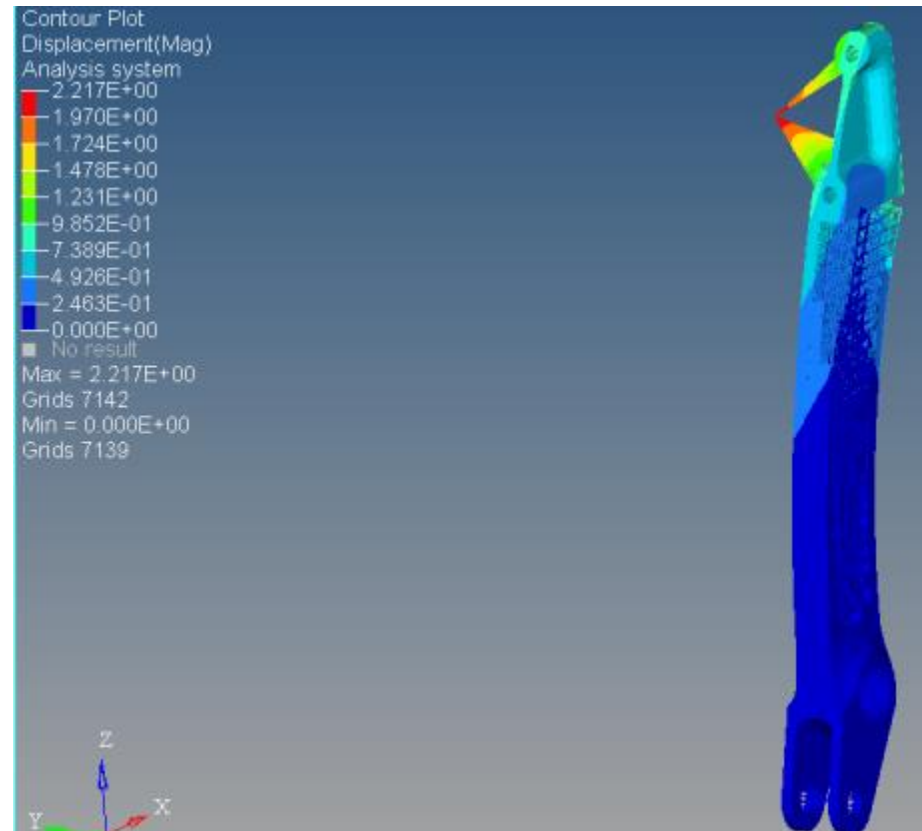
FINAL PART ANALYSIS WAS CONDUCTED ON THE "BEAM" VERSION AND NOT THE FINAL STL.

LATTICE OPTIMIZED DESIGN SPECS

218.0 g

2.217 mm

28% STIFFER, 3% LIGHTER



QUESTIONS?

THANK YOU